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AVIATION

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VOLUME VIII

Number 6

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ABSOLUTE PRESSURE GAUGE FOR WIND TUNNEL USE
UNIVERSAL TEST ENGINE
GOURDOU-LESEURRE PURSUIT PLANE
DIRECTIONAL STABILITY OF AIRSHIPS

PUBLISHED SEMI-MONTHLY

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APRIL 15, 1930

AVIATION

AND
AERONAUTICAL ENGINEERING

VOL. VIII, NO. 6

Member of the Audit Bureau of Circulations

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WYMAN
GORDON

AEROPLANE CRANKSHAFTS

WYMAN-GORDON COMPANY

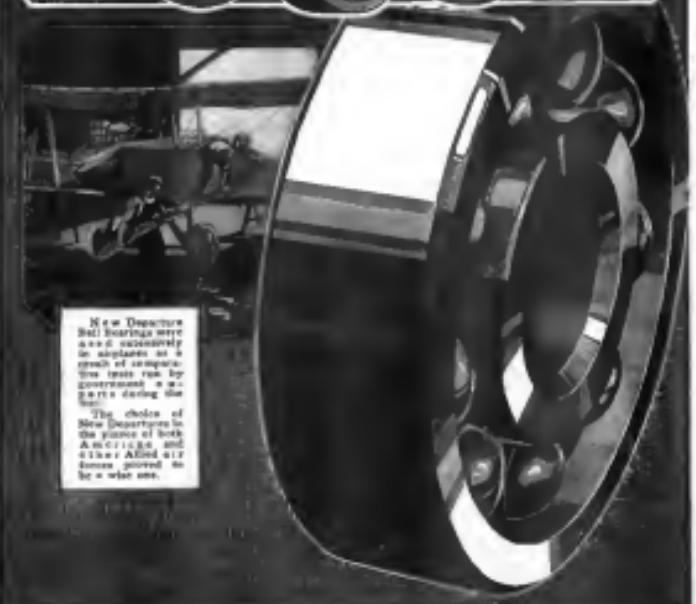
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NEW DEPARTURE

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AVIATION

AND

AERONAUTICAL ENGINEERING

Vol. VIII

April 15, 1928

No. 6

THE Aeronautical Show which opens on April 21, at San Francisco at the first affair of this kind ever held on the Pacific Coast. As such it will undoubtedly attract a large number of visitors whose practical acquaintance with aeronautics was hitherto restricted to seeing various Army airplanes in the sky or on the ground. A convincing demonstration of the stage aviation has now reached in its application to the demands of air transport was hitherto lacking in their important section of the United States. The coming Aeronautical Show will fortunately correct that deficiency and the Manufacturers Aircraft Association—to whose untiring efforts the exposition will owe its existence—will be congratulated for having taken this step, which is in keeping with its far-sighted policy.

It is therefore to be hoped that the Show will do much to arouse the Pacific Coast in the realization that aircraft offers a means of communication superior to any other in the matter of speed, and fully comparable as to safety and comfort.

Indication of Wings on Sand Test

At the present moment there is a feeling amongst aeronautics, that the commonly used method of 14 deg. on sand testing is excessive, and that 7 or 8 deg. would be a more correct figure.

When the 14 deg. indication was first employed in this country, the present figure was fixed on no very sound theoretical basis, but was chosen to conform with French practice, and to impose a heavy drag load on the wings, as drag loads seemed to be the cause of the most serious trouble in flight.

The indication of 14 deg., however, tends to impose a greater drag on the wings than would be present even on the steepest of dives at terminal velocity, and a much greater drag load than is present on recovery after a dive.

It would seem as if decreasing the indication of the wings on sand tests would be a step in the right direction.

Maintenance of Flight in Twin Engine Machines

Recent experimentation on propellers indicates that a propeller, whether it revs. or rotates slowly in a wind tunnel, may have quite a considerable negative thrust coefficient. This negative thrust coefficient may be in the neighborhood of 20 per cent of the thrust coefficient of the propeller under ordinary flight conditions.

This result is not unexpected by engineers, and has an important bearing on the question of maintenance

of flight for the twin engine machines with one motor out of commission.

It has sometimes been thought that in order to determine the possibility of flight on one motor, all that was necessary was to draw the usual required horse-power curve, and tell the horse-power available. Unfortunately, that is much too favorable an assumption.

With one motor, there is considerable rudder control required when in turn means banking, and consequent loss of sustention. If the motor is just turning over and flying, its thrust coefficient may be zero, but if it is completely at rest or revolving slowly in a windmill, it will produce a large negative thrust, which will result in an appreciable decrease of available power.

This point requires very careful consideration in the design of twin engine planes and in testing them if would seem particularly important to have one motor completely out of commission, and not revolving at a low speed as has been customary.

Radial Air-Cooled Motors

The recent specifications issued by the Engineering Division of the Air Service, mark a very important development. Although the stationary air-cooled motor has not yet got beyond the experimental stage, there is no doubt that such engines are destined to play a very important part in all future airplane development.

The Air Service, while no doubt realizing to the full extent the success that the British constructors have achieved in this field, is very wisely calling on the best American talent, so that the problem may be tackled independently on this side.

The specifications call for a very high standard of achievement, but, at the same time, give the designer great independence of action. A horse horsepower of between 350 to 370 is called for at 1,800 r.p.m. of the motor. This probably is as good a coordination as the aeronautical designer can hope to have. The fuel consumption called for must not exceed 50 lb. per h. b.p. hour, which is a moderately liberal figure. The engine must not weigh over 2 lb. per h. b.p. at normal r.p.m. The maximum diameter is not to exceed 50 in. The time imposed are extremely severe, and include a 60-hr. endurance test.

Builders are given plenty of time for the development of the engine and no doubt recognition will be on a liberal scale.

It is extremely gratifying to see this important development started by the government on such satisfactory lines.

ALBERT H. KELLOGG
PRESIDENT AND CHIEF
CHARLES DUNSTY
EXECUTIVE SECRETARY
EDWARD FOWELL
GENERAL MANAGER

Universal Test Engine

By Glenn D. Angle

Engines in Charge of Engine Room, Engineering Division, the former

Engineers, will generally agree, that despite the marvelous progress made in the development of high speed internal combustion engines during recent years, there still remains much to be done in the direction of increasing the power output per pound of engine weight. The demand for better motors places a heavy premium on power per pound. Both new ideas affecting improvement in weight, and new ideas affecting improvement in power output, are being constantly developed, and some of the more recent developments, which could be used in developing a power plant for transoceanic flight, are described below.

Nevertheless, the responsiveness of a cylinder has not been cast on a complete sugar, but sugar produces indistinct results as quickly as desired. Moreover, for multi-cylinder

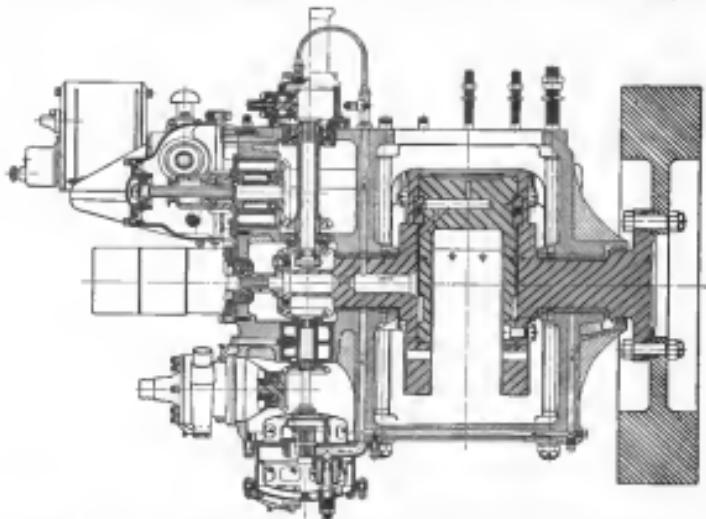


FIG. 1. Lissocampus. Cross-section of Terminal Test Tissue.

individual analysis of each source is potentially impossible. Furthermore, engine designs are usually of such a varied nature that only the most pronounced differences in performance will permit of conclusions regarding the effects of certain solutions. In addition, something should be done as far as the lack of substantiation data is concerned. A sensible program has probably been the engine as a quantity in itself in making the design or substantiation of designs that should produce the results specified in a more efficient manner.

As a rule, the mechanical efficiency of an internal combustion engine is comparatively high. This is particularly true of the

April 16, 1968

A YIAO TIE

versal Test. Engine surmounts this difficulty, as it is so designed that main and cylinder can be twisted on it, but before describing this engine it would be well to briefly explain the specific reasons for its design.

Immediately following the signing of the armistice, the organization of the Governmental Experimental Airplane Station at McClellan Field, Dayton, Ohio, began to rapidly adjust itself to the new duties, which included principally the perfecting of certain airplanes and engines that are on hand, and as a technical division to accumulate data and prepare designs for future construction which should place this country in its proper rank in the absence of military armistice.

Slow engine development should provide plane development by one year or more, it was clearly evident that a good engine program was of the utmost importance. Only a few American designs were considered of military value and in addition immediately undertaking the necessary improvements on these engines, the design of other types deemed to be needed was also planned under careful consideration.

For future designs it was very apparent that the Bureau methods of development were entirely inadequate, and the data and results need hardly be expected with the limited amount of money available for this work. It was therefore decided to construct some sort of engine, which would allow for accurately testing different sizes and designs of cylinders under varying conditions, so that the performance and consequently the value of any particular design could be determined before



Fig. 8. See Note on Theoretical Test Results.

which occurs when it might be desirable to determine the effects of different compressions at various altitudes by noting a cylinder's maximum altitude chamber. All other outstanding features can best be explained in the description of the various parts which immediately follows.

Conclusion

The framework is made in halves and divided along the central crack center-line in the conventional manner. Both upper and lower covers are crisscrossed. Bearings and pins are held together mostly by four long and sturdy wing nuts, together with the four shorter bearing bolts. These are also additional holes through the outer flanges which are all tight gauge. The casting, when it is made of steel, produces a very rigid frame member and should prevent lateral bending or shear due to lateral wind loads.

The road and gear compartments are separated by a wall. The gear are not exposed in the roadster top. On either side of the road compartment are large head lamps, front parcel box and the adjustment of steering radius. Where the engine is in running condition these supports are removed by taking off the retaining screws and the supports are then removed to prevent oil vapour from being blown out or any such substance from vapourising in the engine.

Granular

constructing an entire engine. Furthermore, the Engineering Division could become acquainted with the exact performance of any cylinder selected by the manufacturer of an experimental engine prior to purchasing the complete unit.

No knowledge whatever, any formal construction of this sort existed; consequently, the practicability of building as required by these particular conditions was not known. It was not until the entire situation had been very carefully analyzed that design and construction was actually undertaken. It was proposed to obtain, as wide a range of hook-strike combinations as practically possible, provide for the operation of all kinds of gates and valve mechanisms, and at the same time allow complete interchangeability of parts and easy access for inspection.

The salient features of the engine as it was finally worked out in the wide range of cylinder adaptation. Cylinders can be graduated from 4 to 100, inclusive from 4 to 100, inclusive, and engines from 4 to 1000. The engine is built in two sizes of sizes of airplane engines that are at present in practical use, and allows for experimentation with large cylinders if future requirements as demanded. Compression ratios can be readily varied from 6 to 10 by taking cylinder space to obtain a ratio from 6 to 10. Obviously, the engine can be easily and conveniently made to generate heat, pressure, and power for



FIG. 4. Upper Half of Chart.



Fig. 5. LOWER HALF OF CRANKCASE

have been made for each length of stroke, but that undoubtedly would have introduced difficulties later on. Bearings would be required to support the rod, and the rod would have to be supported in a conventional shaft design, in order to be strong enough for the large bore-stroke construction. The body of the rod is not long enough to support the weight of the connecting rod, so the bearing required to support the extremes within safe limits will be the largest diameter cylinders which were likely to be used, and also near the shaft of the smallest cylinders, whose diameters and case would not be less than the stroke length.

Since the connecting rod is more changed, the crankpin bearing is also more affected, and the bearing has to be designed to satisfy all conditions. The upper end of the rod is therefore made simply large and a bearing inserted to give the desired diameter of piston pin bearing. The width of the upper and of the lower bearing is the same, bearing being made in the same position that would be required with each rod, and by careful analysis it was found possible to so proportion these bearings that at no time were potential shaft-bearing pressures exceeded.

Connecting Rods

It was necessary to provide 100 shafts in order to test all types of L or T head, or in fact any cylinder design, since there were 100 specimens. Separate crank pins have a crankshaft assembly which is bolted to the crankcase, and the connecting rod is bolted to each side of the case, and at a distance from the center distance, which would be used. The shafts are driven through sleeve gears, as will be noted from Fig. 2. An idle gear is provided on a sleeve which carries the driving and driven gears, and also a sleeve which carries the driving gear on the shaft which it drives or not to be used. The crankshaft gear is used to engage on the shaft by these sleeves. By providing a certain different odd number of holes at each point, a very fine degree of adjustment for timing purposes is had.

The crankshaft assembly consists mainly of three major parts. The two end portions include the main journals, generally situated in the taper down which have necessary recesses for receiving the intermediate plates. One end plate has a flange for attaching a flywheel and the other a flange for a level point which drives the various auxiliary mechanisms. The intermediate plates consist of a series of webs, not necessarily formed similar and integral with the plates, the plates having a diameter of 10 inches to the recesses provided in the larger shafts of the end sections. The eccentricity of the plates is the sum of the maximum and minimum crank radii required to obtain the required engine speed. By angular adjustment of the intermediate sections it is clearly evident that varying crank radii can be readily secured.

The three crank sections are held together as one assembly by a total of 10 bolts, 6 on each side. These bolts have counter-bore heads, are in flush to the outer surfaces so as to avoid any interference with the connecting rod, and are retained by slotted lock washers. The connecting rod is held in the crankcase by driving torque is carried by blocks or webs which fit snugly into small recesses on both sides. To facilitate removal, each web is provided with a tapped hole.

Coupling webs are bolted to the crankcase side of the large side of the end sections by means of bolts provided in these portions outside the eccentric recesses. The bolts are so arranged that they are capable of being removed without any appreciable interference from some permanent location. The weights are of such proportions and the centers of gravity are so located from the crank center, that the crankshaft assembly in addition to being in static balance provides for the balance of half of the connecting rod. Hence, as is generally understood, perfect balance of the crank is not required. The balance can be so accomplished only at the expense of introducing an unbalanced component of equal magnitude along the longitudinal. However, as a result, the forces are least if considered in every direction.

Camshaft

It became necessary to provide for the cams so that a piston can overrun the cylinder skirt, but it was found that only one rod was passed for each even stroke length. The body of the rod is not long enough to support the weight of the connecting rod, so the bearing required to support the extremes within safe limits will be the largest diameter cylinders which were likely to be used, and also near the shaft of the smallest cylinders, whose diameters and case would not be less than the stroke length.

Since the connecting rod is more changed, the crankpin bearing is also more affected, and the bearing has to be designed to satisfy all conditions. The upper end of the rod is therefore made simply large and a bearing inserted to give the desired diameter of piston pin bearing. The width of the upper and of the lower bearing is the same, bearing being made in the same position that would be required with each rod, and by careful analysis it was found possible to so proportion these bearings that at no time were potential shaft-bearing pressures exceeded.

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Overhead camshafts are to be considered as a part of the cylinder design, and therefore need no descriptive term. However, attention should be directed to the fact that the intake and exhaust cams in sizes carried out, even as far as to use the same size of plates, then making it possible to adapt the same cams in other place.

Flywheel

The purpose of the flywheel is quite generally known in as far as the function of maintaining a lower motor speed, and hence, the result of storing the energy of the engine in pounds of high inertia to compensate for the low torque periods. We see in this case the possibility of compensating high and low torque magnitudes and also a wide speed range, if it is possible to accomplish by the so-called journal bearing. A flywheel of this type is not a motor, but is a device to handle the greatest torque periods, of course very little, if any, would undoubtedly have productive peripheral speeds in certain cases. It was found possible, however, after several trial computations, to satisfy all conditions with only two sizes of flywheels. These were so designed, that at no time would the engine be subjected to precession from rotating at its lowest speeds more than 300 rpm.

Camshafts

Lubrication is of the forced feed dry sump type, oil pressure being maintained by a slightly altered Liberty oil pump mounted on the lower part of the crankcase. The only alteration on the pump is the addition of an exterior support for varying the pressure of the oiler valve spring. This provides means for regulating the oil pressure which is maintained in the bearing as well as the shaft. The oil pressure and valve connections were incorporated so that oil could be fed to any one or all of the shafts. The bearing pressure, the gears and bearings used in conjunction with driving all of the various gears and also to an oil gauge.

Gauge

Either battery or magnetic ignition may be employed. A Liberty generator is supported on the crankcase and driven through a belt and pulley. The generator is of the standard type and is to be replaced by a current one which fits the mounting stage. Lubrication for battery ignition is taken care of by magnetic replacement units mounted on the magnetic base plate.

The rear case cover provides space for mounting and driving four magneto or four magnetic replacement units as desired. It is possible to locate the cylinder here dimensions are increased the use of four spark plug per cylinder should



Fig. 6. TWO BENZ-Tv CRANKCASES

improve performance as a result of the better flame propagation. By supplying at least four spark boxes per cylinder, it will be possible to vary the ignition values of the different numbers of cylinders. Also, the proper location of spark plug leads and plug cylinder sleeves may be accomplished by making the connection to the merits of the merits of the two systems under different conditions. It is a very good and manner in which that engine to run tests to determine the values of various ignitions of spark.

In order to derive full benefit from multiple ignition, it becomes important to have the spark units synchronized. All ignition, therefore, should be advanced together from a central point. The magnetic ignition is synchronized by a clutchless gear from cross shafts whose squared ends engage with a central and permanently mounted shaft having an integral speed gear. The angular position of the drive as in one shaft is varied by shifting the driving central gear forward and back and the driving gear is so mounted that it can be rotated by a yoke which is pivoted to a small shaft extending in conjunction with the housing for took up as an adjustment lever.

The cross shafts on either side of the center can be readily removed from the ends by first disengaging the covers which reveals their ball bearings. This feature is important as two main and gear assemblies are supplied in order to operate magnetic bearing either crankshaft or half crankshaft of speed.

It will be noticed from the interior view of the magnetic bearing shown on Fig. 7 that special means have been provided for accurately adjusting the magnetic bearing to the spark system. The magnetic bearing is not fastened to the exterior shaft, but instead, it is held in a sleeve which is fastened to the shaft. The sleeve is so screwed up and locked in the correct position against an extension provided on the back of the gear.

When certain magnetic or magnetic replacement units are used, the side through which these units are driven must be held in a special place. This place is held in position by a flat spring which after being fastened back out of the way as a vertical position.

Cooling

For testing water-cooled cylinders, circulation is maintained by a rectangular water pump identical to the one designed for the 4-cyl. Liberty engine. The outlet water from the pump is passed just beyond the first external cylinder ribs, and flows between the cylinder and the cylinder wall to any desired point on a cylinder. In this manner it will be found that the capacity of this pump is greater than necessary, so as to provide for the circulation of water when a cylinder is tested as a part of a multi-cylinder form, instead of replacing and removing them one at a time.

When testing air-cooled cylinders, the water pump should be temporarily replaced by a special cover plate designed for this purpose. An air blast produced by a outside blower is then directed against the cylinder walls for cooling. If suitable equipment is available, some very valuable data in regards to the cooling can be obtained in connection with regular cylinder tests.



Fig. 7. TWO BENZ-Tv CRANKCASES

Turbo-super Divs.

Whatever the dynamometer testing equipment does not require a tachometer, one may be attached and driven from the engine. The driving mechanism is incorporated at the end of the gear and driving shaft shown mounted in the small housing on top of the crankcase directly above the gear compartment. The housing has three possible positions, allowing for drive forward and out of the engine, as shown in Fig. 1, or forward and out as all former calculations seemed to warrant.

Finally all computations will have to be made on a basis of the type of cylinder construction and then if desired, various cylinders measured for the purpose of determining the best combination of these parts for best performance. In a general way, cylinders must be classified and their performance judged accordingly.

Since the mechanical efficiency of this engine is low and brake loads are set in the current practice to those of multi-cylinder forms, it becomes necessary to reduce the data to obtain the best performance with all of the engine parts within these computations. After taking the allowances for friction, the brake horsepower of any combination of cylinders may be approximately obtained by totaling the indicated horsepower of these same cylinders and multiplying by the gross mechanical efficiency of the engine. This is the best estimate of the same type results at corresponding speeds.

One of the design factors which engineers seldom agree upon, is the interlock valve. It should be possible after a sufficient number of combustions have been made on this engine to determine load of cylinder, statement to the regard. Cylinder intake valves are a matter of a question of the type of cylinder and the work to be performed, nevertheless this should be reasonably determined. For every case, the valve and cam change, which continually requires extensive experimenting can be easily worked out with full assurance that the cams adopted are the best possible for the engine. Ignition is another particular factor to consider and location of spark plug, ease simple to perform and the results should be conclusive. Tests on cooling, whether by means of water or air, can be satisfactorily conducted and the data made applicable to the engine of which the cylinder intake, intake and exhaust valves are the same as the apparently very light development work on cylinders which should be completely and accurately accomplished on the engine.

Two test engines were constructed in the shop at McCook Field and sufficiently well built already have made on a few tests. The first engine, in making power output of the same and when a test is finished, that engine is removed from the dyno, measured for a new cylinder adaptation, and the other engine takes its place. In order of efficient passing it is possible to so arrange the order of these tests that very little work is required in changing cylinders.

Several cylinder heads have been completed and are ready for test but with a number of others are in course of construction. The results so obtained, however, can be used as to the practicability of this procedure, either in developing the cylinder or obtaining useful data on cylinder design generally. As soon as these are obtained, the writer will be in a position to compare the data available and arrive at a conclusion which at least may have considerable influence on cylinder design to the future.

Compressor Ratio

As heretofore stated, compression ratios from 4 to 18 are possible. The ratios are varied by raising or lowering the cylinder in respect to the crank case, and, since there are two, a suitable balance must be obtained. The crankcase has a large hole given the crank compartment in order to adapt cylinder of 8 in. bore. As a result, each cylinder must be securely mounted as a special design which fits the crankcase, and is held down by means of the long main bearing which is of an additional width of smaller diameter. All of these parts extend above the frame for some distance to allow for cleaning.

It was found possible to obtain the desired range of compression ratios by compressing small increments, by the use of 1 or more of the six mounting plates of different thicknesses which are bolted to the cylinder base. These plates, which have the same slope as the cylinder design, are fitted over the ends before the gauge is put into place. Calculations are always made to determine the compression ratios that will be had with the various stem combinations when used with any cylinder selection and are recorded for reference purposes during the tests.

The above discussions dealing with certain features of the Universal Test Engine explain fairly well the successive pos-



FIG. 1. UNIVERSAL TEST ENGINE. MAGNETIC BRAKE, OIL PUMP, WATER PUMP AND GENERATOR

rated for this cover whenever overhead camshafts are used. Oil is then supplied to the gear compartment by the injection into the camshaft housing.

Overhead Camshaft Drive

Overhead camshafts are always driven through bevel gears. The driving pinion is keyed to a vertical shaft, having splines at its lower end, which is held in a bearing mounted in the base of the case. There is no question of vertical camshaft drive-shafts are all that are needed, to take care of the variations in height of camshaft above the top of the crankcase as influenced by cylinder design and by differences in base stroke, eccentric and compression ratio. Since the camshaft housing is mounted on the base, no camshaft drive-shaft need be made separately. This allows almost unlimited possibilities in the matter of raising or lowering the cylinder for any purpose.

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Absolute Pressure Gauge for Wind Tunnel Use

By Dr. J. G. Coffin

Director of Research, Curtis Aerodynamic Laboratories

In the Curtis wind tunnels the air speed determinations are based upon the readings produced by a standard Pitot tube as an absolute pressure gauge of great simplicity. This gauge replaces the well known Chattock gauge, a sketch of which is shown later for comparison.

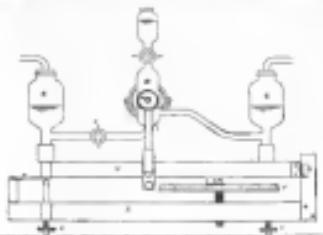


FIG. 1. SKETCH OF THE CHATTOCK MANOMETER

A description of the Chattock gauge by J. G. Ellsworth is available in the "Reports on Wind Tunnel Measurements in Aerodynamics," Bureau of Standards Collection, Vol. 62, No. 4. In our opinion, this gauge suffers from three disadvantages, one of which is that it is too sensitive when air speed is high. Another is that it requires a manometer tube in which air speed is measured which is tilted through large angles, and another is that it requires three different liquids for use and must be put into working condition area after area. It has been used for 20 days. Practically no velocity at high speed can be measured accurately to 10 percent in practical use. Our Chattock gauge is in accurate at air speed of 45 ft. per sec. It shows a change of air speed of 0.03 m.p.h. and an 80 ft. per sec. a change of 0.64 m.p.h. This is an unusual finding since the errors are 10 times as great. It is only just to state that Dr. Ellsworth has done the impossible for the present instrument at very small pressures. As an attempt to correct this instrument would be to connect A and C by means of a flexible tube, and supply man. C vertically, keeping A and D fixed. This would eliminate the second of the above objections, but not the third.

The Chattock gauge is a manometer, however, from some of these disadvantages it is always in working order and is simply necessary for most wind tunnel use. Two photographic views are shown in Fig. 2 and Fig. 3. The instrument is shown connected for use with the standard Pitot tube, which can be seen in the Fig. 2 and 3 are scale drawings of the gauge.

In Fig. 2 it is a manometer base and A, B, C are three wooden posts to support the glass reservoir D. The dimensions of this reservoir are so large that the change of level for any variation of the liquid in the glass tube G is inappreciable. As a matter of fact, when the liquid reservoir D is empty, the height of the reservoir above the base of the gauge is air ready for this pressure. However, as the gauge might be used at times allowing the liquid to move up, it is convenient to have a large reservoir.

The reservoir D is provided with two outlets C on top in

¹ Description of the Chattock gauge, 1931, Bureau of Standards, Vol. 62, No. 4.

the center and D at the bottom on the side. The long outlet is connected to the lower end of an inclined glass tube H which is attached rigidly to a manometer tube I. This tube can be rotated about the axis ED and be held in any position by the stop J. By changing the inclination of the tube H the air pressure in the glass tube G can be measured. The static terminal of the Pitot tube is connected to the upper end of the glass tube G and the dynamic terminal to C.

The accuracy of the Chattock depends upon the care given to the glass tube on the outlet end I. This end, which is turned rigidly into the glass tube plate, must have a sufficient number of accurately ground holes, in our laboratory, at exactly half-inch intervals. In order to insure accuracy in the spacing of these holes, steel gauge rings were made, two of which had a hole in them, so as to be used as a gauge for the holes. By using several of these rings of different lengths accurately known, it was possible to drill and size the holes in these holes. They are accurately placed to less than a thousandth of an inch, which is ample for our purpose.

The two glass tubes H and I are fitted to slide without play on the base. The base is made of wood and is provided with a slot running along a plane P, but can not freely rotate through a distance of about two inches.



FIG. 2. CURTIS ABSOLUTE GAUGE AND PITOT TUBE

The lower one consists of two parts H and I, and is mounted in a manometer. In lower part H, is rapidly held at any convenient height by means of the plug F, the upper part can be raised or lowered by rotating through a range of about three-quarters of an inch by turning M. There are four holes in the base of the tube H and the head being divided into four quadrants. The manometer tube I is connected to the gauge H.

The height of the reservoir D above the level of the liquid in the manometer can then be accurately determined. The remaining quantity necessary for the absolute determination of the air speed is the density of the liquid used. We have found that water, which is to be very satisfactory. The density of the liquid is determined for several temperatures around a normal value and used in the computations.

The Air Service of the United States Army and Navy will be represented by a very comprehensive exhibit.



FIG. 3. CURTISS ABSOLUTE GAUGE, READING BAROMETRIC PRESSURE

With the wind off, the slides are adjusted until the aneroid M comes at the center of the glass tube G , and at such a position that a rotation of the dial does not alter the reading. The effect then of giving different accelerations to the dial, and therefore of the gauge tube, is neutral to alter the reading. By making the angle with the horizontal very small, the instrument becomes very sensitive.

One great advantage of this instrument is that as the readings are always at M , the effects of changes of base of tube and capillary are entirely eliminated. Dr. Hanauer found these effects to be serious, errors amounting to several per cent being

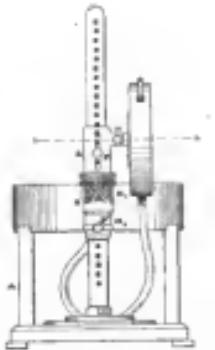


FIG. 4. CURTISS ABSOLUTE PYROMETER GLAUGE

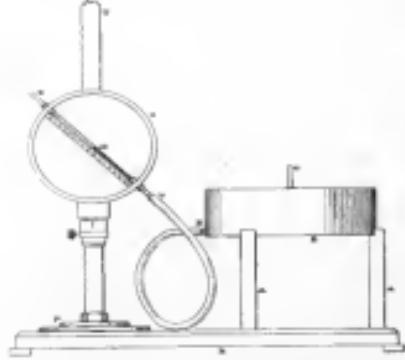


FIG. 5. SIDE VIEW OF THE CURTISS ABSOLUTE PYROMETER GLAUGE AND DETAIL OF MAGNIFYING SLIDE

common. The enclosed gauge indications are usually too low. He also states that under certain conditions it can be used as an absolute instrument to an accuracy of 1.5 per cent, but that it is as sensitive to less than 10 per cent changes in velocity as may be introduced by a rotation of the dial, and that when the instrument is used as a barometer it is not as accurate as the standard.

The Curtiss gauge is an enclosed gauge and the "barber manometer" considered. An aneroid above, in actual use there is no change of level in the reservoir as no liquid passes through the reservoir and the glass tube, any amount of damping being obtained.

The method of use employed at the Curtiss Laboratories is as follows: Wind speeds corresponding to every half-inch of level are computed. The height of the dial is adjusted by means of the slide and the view window shows until the liquid meniscus is at the center. The constant air stream passes at maximum velocity through the tube, without any friction loss in the manometer. This height corresponds, of course, to a calculated air speed. The readings of the final enclosed tube never gauge are taken for these speeds and from these the readings corresponding to 20, 30, 40, 50, 60, 70, 80, etc., are found from a table. If, on, the contrary, the air speed in the standard gauge at 100 ft. is measured directly to 20 or 30 or p , the gauge at 100 ft. must be found to be proportional.

The merit of this instrument lies in its simplicity. There is nothing to get out of order. There are no readings as such to be made, the reservoir level does not change, there are no computations due to heat absorption or required corrections, and the instrument is ready for use. We need not use the end F of the rubber tube DF is placed over the outlet C which prevents all losses by evaporation and consequent changes in density.

The gauge can be adapted for very high wind speeds by employing necessary amounts of alcohol or water. The only change necessary would be in using an iron glass or steel reservoir instead of borosilicate.

The differences in pressure (Δp) between the static and dynamic openings of the standard Pitot are related to the speed of the air (V) and the air density (ρ) by the following formula:

$$\Delta p = \frac{1}{2} \rho V^2$$

The Curtiss Aeronomical Laboratory has adopted a standard

air density of .001229 gm./cu. in., which is also in use at the National Physical Laboratory in England, at the Bureau of Standards at Washington and by the Washington Navy Yard. The density corresponds to a value of ρ of 990.0 cu. in./cu. in., barometric pressure of 1000.000. The value of ρ at 0 deg. Celsius is 1.225 gm./cu. in. and at 100 deg. Celsius 0.896 gm./cu. in. With this value one mole of water at 0 deg. Celsius has a density of 1.000 gm./cu. in. and 1000 cu. in. per mole. The maximum density corresponds to an air speed of 65.185 m. p. h. The following table gives values of air speeds corresponding to various readings on the absolute gauge using a fluid of density 2 gm./cu. in.

TABLE SHOWING AIR SPEEDS IN CURTISS ABSOLUTE PYROMETER GLAUGE IN TERMS OF WATER DENSITY

Section	Barometric Air Speed	British Standard		British Air Speed
		Water M. P. H.	Water	
1.	0.00	0.00	0.00	0.00
2.	100.00	0.00	100.00	100.00
3.	100.00	0.00	100.00	100.00
4.	100.00	0.00	100.00	100.00
5.	100.00	0.00	100.00	100.00
6.	100.00	0.00	100.00	100.00
7.	100.00	0.00	100.00	100.00
8.	100.00	0.00	100.00	100.00
9.	100.00	0.00	100.00	100.00
10.	100.00	0.00	100.00	100.00
11.	100.00	0.00	100.00	100.00
12.	100.00	0.00	100.00	100.00
13.	100.00	0.00	100.00	100.00
14.	100.00	0.00	100.00	100.00
15.	100.00	0.00	100.00	100.00
16.	100.00	0.00	100.00	100.00
17.	100.00	0.00	100.00	100.00
18.	100.00	0.00	100.00	100.00
19.	100.00	0.00	100.00	100.00
20.	100.00	0.00	100.00	100.00
21.	100.00	0.00	100.00	100.00
22.	100.00	0.00	100.00	100.00
23.	100.00	0.00	100.00	100.00
24.	100.00	0.00	100.00	100.00
25.	100.00	0.00	100.00	100.00
26.	100.00	0.00	100.00	100.00
27.	100.00	0.00	100.00	100.00
28.	100.00	0.00	100.00	100.00
29.	100.00	0.00	100.00	100.00
30.	100.00	0.00	100.00	100.00
31.	100.00	0.00	100.00	100.00
32.	100.00	0.00	100.00	100.00
33.	100.00	0.00	100.00	100.00
34.	100.00	0.00	100.00	100.00
35.	100.00	0.00	100.00	100.00
36.	100.00	0.00	100.00	100.00
37.	100.00	0.00	100.00	100.00
38.	100.00	0.00	100.00	100.00
39.	100.00	0.00	100.00	100.00
40.	100.00	0.00	100.00	100.00
41.	100.00	0.00	100.00	100.00
42.	100.00	0.00	100.00	100.00
43.	100.00	0.00	100.00	100.00
44.	100.00	0.00	100.00	100.00
45.	100.00	0.00	100.00	100.00
46.	100.00	0.00	100.00	100.00
47.	100.00	0.00	100.00	100.00
48.	100.00	0.00	100.00	100.00
49.	100.00	0.00	100.00	100.00
50.	100.00	0.00	100.00	100.00
51.	100.00	0.00	100.00	100.00
52.	100.00	0.00	100.00	100.00
53.	100.00	0.00	100.00	100.00
54.	100.00	0.00	100.00	100.00
55.	100.00	0.00	100.00	100.00
56.	100.00	0.00	100.00	100.00
57.	100.00	0.00	100.00	100.00
58.	100.00	0.00	100.00	100.00
59.	100.00	0.00	100.00	100.00
60.	100.00	0.00	100.00	100.00
61.	100.00	0.00	100.00	100.00
62.	100.00	0.00	100.00	100.00
63.	100.00	0.00	100.00	100.00
64.	100.00	0.00	100.00	100.00
65.	100.00	0.00	100.00	100.00
66.	100.00	0.00	100.00	100.00
67.	100.00	0.00	100.00	100.00
68.	100.00	0.00	100.00	100.00
69.	100.00	0.00	100.00	100.00
70.	100.00	0.00	100.00	100.00
71.	100.00	0.00	100.00	100.00
72.	100.00	0.00	100.00	100.00
73.	100.00	0.00	100.00	100.00
74.	100.00	0.00	100.00	100.00
75.	100.00	0.00	100.00	100.00
76.	100.00	0.00	100.00	100.00
77.	100.00	0.00	100.00	100.00
78.	100.00	0.00	100.00	100.00
79.	100.00	0.00	100.00	100.00
80.	100.00	0.00	100.00	100.00
81.	100.00	0.00	100.00	100.00
82.	100.00	0.00	100.00	100.00
83.	100.00	0.00	100.00	100.00
84.	100.00	0.00	100.00	100.00
85.	100.00	0.00	100.00	100.00
86.	100.00	0.00	100.00	100.00
87.	100.00	0.00	100.00	100.00
88.	100.00	0.00	100.00	100.00
89.	100.00	0.00	100.00	100.00
90.	100.00	0.00	100.00	100.00
91.	100.00	0.00	100.00	100.00
92.	100.00	0.00	100.00	100.00
93.	100.00	0.00	100.00	100.00
94.	100.00	0.00	100.00	100.00
95.	100.00	0.00	100.00	100.00
96.	100.00	0.00	100.00	100.00
97.	100.00	0.00	100.00	100.00
98.	100.00	0.00	100.00	100.00
99.	100.00	0.00	100.00	100.00
100.	100.00	0.00	100.00	100.00

If the density changes, either through temperature change or by the employment of a different liquid, these values of air speed are to be multiplied by the square root of the new density reduced to water at 4 deg. Cent.

It is hoped that all aerodynamic laboratories will adopt these standard conditions so as to eliminate confusion and discrepancies in experimental results.

The following table is given to aid in the use of the standard aerodynamic laboratories. It will be noticed that although the standard variations of temperature and humidity prevail, the values of the standard air density are given as well as the corresponding values of the greater part of aerodynamic work of the present time, so that for all practical purposes results are readily comparable.

It is strongly recommended to find six out of the eleven laboratories listed are in absolute agreement.

It is to be hoped that all laboratories will use their way to adopt the standard air density of 1000 cu. ft./lb. and standard variations of temperature and humidity, as given in the table.

The variation of the air density varies with the pressure.

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New Pursuit Ship-Plane Used by the United States Navy



THIS AIRCRAFT TAKES OFF ON ITS WHEELS FROM BOARD SHIP AND ALIGHT ON THE SEA, WHERE THE AIR BAG KEEPS IT AFLOAT
U.S. NAVY PHOTO, FROM ENTITLED REPORT

Special maps are located on the ends of the central cables to permit quick assembly and dismantling of the ship.

Instrument Board.—The main idea carried out in the instrument board is to have as many instruments as possible on the left hand side of the board and the balance concentrated on the right hand side, lighting all by one electric light. The most needed instrument, the manometer, is located in the center. The following instruments are mounted on the board: Tachometer, manometer (by special arrangement the manometer also serves as an altimeter), aeronautical weather station, a set of pressure and temperature gauges. All the instruments are incandescent lighted.

The manometer mentioned above is of standard liquid type, the same as previously used in all Goodyear ships. This manometer has been specially modified to a point where it can serve to indicate altitude as a normal position when ascending or descending, thereby giving more accurate readings than can be obtained if the altitude is read with the ship's altimeter. To take advantage of the leveling point, a pointer is also attached which indicates on a dial in degrees the angle of the ship.

Navigation.—There are cable telephones along the front of the ship to be easily removed by taking out a rivet or two on the outside. The idea is to have a means for quick assembly and dismantling of the ship. Considerable thought was given to this suspension so as not to have any interference with passengers or the ship's equipment.

Top.—For protection in cold weather a light top is mounted over the cockpit. A triple glass windshield is used to enable the pilot to have a clear vision. The side curtains are so constructed that they immediately open by means of a rubber band when the pilot is about to take off. The side curtains are put out of the way very quickly in case of emergency. The top is quickly detached from the rear body when not needed.

Pavilion.—The curtains are both out and inside of the rear and in the rear seat; they open through the bottom of the rear seat. A padded seat is placed in each side of the rear seat and is held in place by a double leather belt on the outside. This seat is fastened to the rear by two cables 6 ft. long, so that when the passengers are seated the pack will fall clear of the rear before it hits the pack.



COURTESY POST BROS. & CO., WITH LAWRENCE 60 HP. RADIAL ENGINE

Drag Fins.—Ninety-five feet of 16-gauge wire is located under the pilot's seat as a suitable container. This drag wire can be released when needed by means of a trap-door on the bottom of the car. When released for use, one end of the wire is fastened to the front seat and a handle is formed by cable which is attached to the rear seat. The arrangement permits easy towing by holding of the wire.

Doors.—Two seats are provided to comfortably accommodate two passengers. The rear seat is, however, sufficiently spacious for two passengers, if desired. The seats are built of slate and upholstered bottom and back, making a comfortable seat for long flights. The pilot's space, including the seat, is very roomy.

Engineering Division Has New Test Furnace

The Material Section, Engineering Division, Air Service, is supervising a test of the Gaseous Furnace at Shreveport, La., which is to be used during the summer of 1928 for the purpose of heat-treating steel taken from the bottom of ships for salvaging purposes. It was not completed until recently. It is now completed and undergoing a series of tests to determine the feasibility of producing heat-treating steel specimens taking of very high tensile strength and elastic limit, to particularly match those of ship plates No. 1020B, for safe taking.

The furnace, located at the end of the Gaseous Section Test Cell No. 7B, as described in *50 Years*, is fully developed and successfully controlled as an integral heat-treating furnace with very narrow limits. The taking is lowered into the furnace, which is made so that the top of the furnace is level with the deck, in a steel enclosure (approximately 1500 ft. of 16-gauge steel) which is 10 ft. high. The furnace is required to heat the specimen with a large amount of heat, transported over the quenching bath, where the bottom of the furnace is exposed and the steel taking, at the quenching temperature, is allowed to drop into the oil quenching bath.

The heating operation is conducted in a similar manner, except that the maximum temperature of the taking in the furnace is of course, lower than it was for the heat-treating operation. The temperature per aggregate is 1000 deg. F. for the quenching operation and 800 deg. to 1000 deg. for the heating operation, depending on the quality of steel used and the physical characteristics desired.

This furnace will be used principally for the heat-treating of steel taking with special reference to the taking and the heating. Up to the present time, all the taking has been produced in quantities of 1000 ft. and the requirements of Specification No. 1020B calling for 200,000 ft. tensile strength with 5 per cent elongation in two inches.

INTERNAL ARRANGEMENT OF THE NEW CAR OF THE POST BROS. AIRSHIP FORWARD



INTERNAL ARRANGEMENT OF THE NEW CAR OF THE POST BROS. AIRSHIP FORWARD

The Gourdou-Leseurre Pursuit Airplane

By Charles Gourdon *

General Fundamentals of the Problem.—We wanted to build a fast monoplane which was to measure up to the Hispano-Suiza 600 hp. engine. We determined on this construction for the following reasons:

The efficiency of the monoplane as compared to that of the biplane is the result of a great many simplifications. Also, consideration of the performance of the Biplane with 100 hp. Hispano-Suiza engine and their close agreement with laboratory tests showed us that it was possible to greatly exceed the results of the tests with a monoplane without increasing the weight of the engine. The biplane was very unstable, especially for the period in which it was built; it has, however, had no day. The wood construction was within the capabilities of a large number of factories in the Paris region, but when it commenced to appear a little slow and when the Technical Section was formed to the use of engines of greater and greater

parts of the machine should in general be calculated with the same indifference.

Choice of the Dimensions of the Machine.—The dimensions and the weight of fuel to be used for two and one-half hours flight, and consequently with existing machines made to approximate 2,470 lbs.

As the loading of 92 lb. per sq. ft. did not appear sufficient for a monoplane we chose an area of 200 sq. ft. to obtain this area in a monoplane without giving it extreme span which would have increased considerably the weight. The aspect ratio of 7.50 was chosen and 0.65 the camber. The aspect ratio (4.7) of such a wing may appear small, but strength considerations made it necessary for us to extend limited dimensions for the span.

Choice of the General Arrangement of Trussing.—Deformations of the wing surface under load is to be avoided as it re-



Front View

power it appeared to us clearly that the risks to be run in constructing a new airplane increased with a new engine and not counterbalanced by the hope of obtaining sensational results.

Experience has confirmed our predictions, since the performances obtained with our surfaces are comparable to those of an machine having engine of 200 hp.

The choice of the engine and monoplane construction being determined, we also wished to make an extremely strong metal machine. Numerous aerobatics with monoplanes have shown that the low total resistance of these airplanes should correspond to very great speeds. Let us assume that the machine has to fly at a speed so that it reaches its maximum speed. The force of gravity is then the angle of the airplane, and in a vertical drop the speed increases and the resistance of the air on the wings, fuselage and propeller counterbalances the effect of gravity. The speed reaches that degree immediately. The speed reaches that degree immediately. The machine is then in equilibrium. The angle of the wings is then constant.

We take into account the moment of our surfaces, and we perceive with notable assistance caused us to predict a driving velocity of about 160 m. p. h. If the pilot raises the machine very quickly when the greatest speed is reached the wings will be highly loaded. We have assumed that the machine will be loaded in one second, as follows, however, the length of time is not limited in practice. Under these conditions it is necessary to assume that the machine supports a load about four times the normal load. This is as easy as possible, apart from defects in construction, the frequency of accidents on monoplanes, the low total resistance of which permits speeds in the neighborhood of 220 m. p. h. and the parts of which could not withstand four times more than five times the normal load. At the beginning of the test the factor 5 was believed by the Technical Section to be ample.

We considered it necessary to design the parts of our machine so that, when they were stressed to their maximum, the elastic limit of the metals employed should not be reached or at least should not be exceeded. We, therefore, preferred for ourselves the condition of having the metals stressed in the neighborhood of their elastic limit when the machine supports four times the normal load. It is understood that all the

shows the efficiency and stability of the machine. A rigid machine cannot be made if cables and plates were used to maintain the structure, as the supports of the flying structure, their strength of service being too great. After the trials and tests can be used only in tension, and while trussing should be decided so that the wing will withstand top loads when flying upside down and standing. We, therefore, decided to employ four wires, suspended from the center of the fuselage. The machine is also suspended so that it has a low head resistance also, the air passes in low on the top of the wing, therefore, by virtue of a well-known theorem, the air speed is greater above the wing than below, and the resistance of parts placed above the wing is less than that placed below.

Each of the wing spans is attached at five points to the rigid trussing which connects the wings to the fuselage. We have chosen the position of the points of attachment on the span so that the bending moment is equal at each of the points. We have also chosen the points of attachment so that the spars are under bending stress by the action of the loads and under compression by reason of the resistance of the whole structure. Computation shows that the bending moments at the point of attachment of the wings are equal if the loads are applied at the center of the span and the resistance of the wings is at the size of the wings. The loads would be obviously equal if the compression due to the angularity of the wings is not taken into account.

The points of attachment of the wings then choose the best position of attachment which is the center of the span, with the angle of the wings to the center of the span. The angle of the wings and the span will be due only to the elongation of the struts under load and to shortening of the span. Variations of length in these parts are exceedingly slight, and we may consider that the variation of the length of the struts is of the order of one-tenth of the span. The attachment of the wings to the wings is formed of soft steel fittings which are held so as to completely surround the span; the strut is attached to the fittings by two large hollow bolts giving ample bearing surface. The struts are held in the wings by two large threaded sleeves which permit adjusting the wing. It is evident that the trussing system is thus composed of triangles ensuring perfect rigidity. Finally, the system of the outer struts are connected by small struts to the points of attachment of the inner struts so that the former will withstand the load of flying upside down with a sufficient factor (12 has been chosen).



THREE-QUARTER FRONT VIEW

The *Frontage*—From *Induktor* (inducers) (the upper biplane) to the front, the lower biplane, assuming the frontwork of the fuselage. As we have a view of all the wings to the fuselage by struts and cables, we must find out which are the best points on the fuselage to attach the struts.

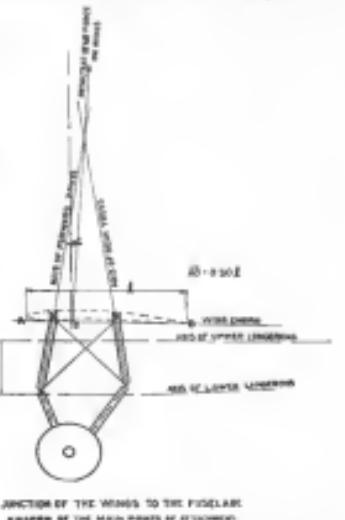
Let us consider a side view of the airplane: the struts, both long and small, and on the fuselage which passes straight out of the wings, open up to the point of attachment on the fuselage. We see that the plane of the fire and air passes down a slight V, the apex of which is directed upward. This arrangement is chosen for the following reasons: We desire to balance the drift stresses on the wing by the strutting (at least in the first flight). We desire to have a small cross-bracing of the wings. The planes of the two bows and struts of struts were prolonged out each other on a straight line parallel to the wing span. In Fig. 5 this straight line is projected to the point of intersection of the two bows of the wings. The point of the rear frame of the wing, which is slightly curved from the point of intersection of the two bows, rises from a point 20/300 of the distance from the leading edge and passes through the point of intersection of the bows of the struts.

As there are several convenient methods of attaining the desired balance we can provide for our own judgment. Let us consider the beginning of the wing, which is the point A. It is assumed that the struts are to sweep slightly aft under the load which it supports. We have, therefore, chosen the position of the struts' attachment so that at this moment all take place, the angle of incidence of the wing is decreased, thereby reducing the angle of attack. This is a diagram on the question of the stability of airplanes on flight.

To be understood better, let us consider the flight of a machine the deformations of which is flight of which are fairly great. This will be the case for example in a machine of large span with cable bracing or in a biplane with large bows with cables. The wings move all along the lines of the struts. In this case the machine will tend to be unstable. Then in a turn, for example, the upper wing having greater speed will move further aft and the angle of incidence will tend to increase, which is necessary for the recovery of the control by the pilot. Even in another case if the angle of the wing to the horizontal is more than 45°, the angle of incidence of the wing will increase, the other from this case its stability is lost so that the load tends to increase. In the contrary, the deformation decreases the angle of incidence, the increase of all leads to lowering the wing to normal conditions. The directional stability of certain machines can be attributed to these deformations which slightly alter the conditions under which the wing operates.



FRONT VIEW



the points of attachment of the wings to the fuselage. On the one hand, on and landing the engine is supported by the fuselage which rests on the landing gear. Finally, the engine support should resist the engine weight with a fairly high factor of safety comparable to that of the entire airplane. The factor 6 seems sufficient.

It is not necessary however to come into the details of the calculations to be made in order to design a proper support. In fact we have seen that the maximum contribution of the airplane makes necessary a load factor of the wings of four times the normal load. In order to carry the fuselage of the engine plus the engine should be capable of withstanding four times the load of the engine. For the required strength the factors we must consider the height of fall (about 1.0 ft. for instance) and the play of the shock absorbers and tires, we can then easily deduce the loads which will be brought to bear on the engine support. We have de-

signed this support by using two diagonal cables attached to the fuselage bungees by several fittings and cross braced by struts. This arrangement with bungees has been adopted by a number of manufacturers.

Outer Portion of Fuselage, Pilot's Cockpit—The fuselage is supported by a combination of a certain number of vertical frames constituted by longitudinal. In order that the fuselage can move however, it is necessary to cross brace the frames. Now it is evident that it will be difficult to place cross bracing in the pilot's position; so the majority of existing airplanes this cross bracing is placed in the rear of the fuselage, which is a standard angle which extends beyond the first frame. During torture tests the fuselage of the *Thessaloniki* airplane was without former produced by a load equal to seven times the normal load on the rear vertical surface. This load was placed in the rear of the fuselage of the above mentioned aircraft. It is assumed that the vertical surfaces can withstand the same load as the wings.

We will not describe in detail the manner in which the tail surfaces, landing gear and other are constructed; we will limit ourselves to say that we observed the following precautions in every case.

The loads of the fittings have been arranged so that the welding is never directly stressed. The attachment of cables or wires traverse or enclose the parts to which they are connected.

In an assembly by bolts or pins these surfaces are such that they can resist shearing stresses, thus their perimeter is sufficient for the metal not to be sheared by shears or otherwise.

We have measured by means of a frequency meter with vibration of the engine, the number of complete cycles of vibration of the machine to particular the engine. We have also measured the number of cycles of vibration of the engine. The rate of vibrations do not correspond to the number of revolutions of the engine at speeds that are employed.

Finally, we have measured the shed support from the parts connected to the engine which are not attached to the skin on taking off and landing.

Chlorophorm—Carried adhering to these paragraphs and considers the importance of which in modern aeronautical construction of today is becoming increasingly evident led us to design an airplane, the characteristics of which are as follows:

Span 30 m. (100 ft.) Length 10 m. (33 ft.) Height 3.5 m. (11 ft.)

Wing area 20 m² (215 ft²) Weight 1,000 kg. (2,200 lb.) Maximum speed 100 km/h. (62 m.p.h.) Maximum altitude 3,000 m. (9,840 ft.)

Flight time 1 hr. 30 min. (100 km. at 60 km/h.) Weight empty 600 kg. (1,320 lb.)

Flight weight 400 kg. (880 lb.) Weight of engine 100 kg. (220 lb.)

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thrust. If the engine can be throttled to a lower speed, the propeller may develop a negative thrust. If the engine is completely dead, and at rest, the propeller will offer resistance instead of the resistance-producing area. Within the engine is dead but the propeller is still rotating, the leading edge of the propeller is moving as a windmill and the thrust may be important, but only when the forward speed is very great, as in a dive.

Let us consider whether any appreciable leading effect can be secured by increasing the angle on the ground at a very low speed. For example, let us consider the case of a low-pitch propeller No. 5 of the W. F. Durand's report No. 14 for the National Advisory Committee for Aeronautics. Assume it is a diameter of 10 ft, and that it is in use with a Liberty engine having a maximum speed of 1780 r.p.m. in level flight near the ground. The $F/\Delta F$ ratio is at a speed of 133 mph, which shows the approximate case of the Curtiss H-4-L machine, $133/1.45 = 93$. In this case V is the airspeed

sped in miles per hour, so the engine speed in revolutions per minute is V/Δ the propeller diameter in feet. The thrust coefficient from Plate VI in Dr. Durand's report is 0.55, and the thrust is given by the formula

$$T = \frac{F \Delta P \rho V^2}{300}, \text{ where}$$

$\Delta = 0.055$, the density of the air.

Applying the formula

$$T = \frac{0.55 \times (1.45) \times (160)^2 \times 0.055}{300} = 360 \text{ lb.}$$

The thrust coefficient is 0.78 so that the torque is

$$Q = \frac{0.78 F \Delta P \rho V^2}{300}$$

$$= \frac{0.78 \times (0.55) \times (160)^2 \times 0.078}{300}$$

$$= 1270 \text{ ft-lb.}$$

The power taken above conditions is 410 hp, which the Liberty engine can just develop. The efficiency is 80 per cent, so that the propeller would be fairly available for the H-4-L Curtiss machine.

In the National Advisory Committee's Report No. 36, the results are given for the same propeller at negative thrust, re-

sponding to 24 hp, however, and if the $F/\Delta F$ ratio were decreased to 1.2 by increasing the engine speed to 405 r.p.m., the negative thrust would still be 137 lb.

The Durand propeller No. 5 was selected quite at random, and yet the maximum possible thrust is found to be 360 lb. at $V = 0$ and $F/\Delta F = 0.55$. This is about 40 lb. more than the skin drag of the same machine, but it is no more negligible, but then propeller is not a very good illustration of what it is possible to do with a propeller in the negative-thrust-producing thrust. In the curves of Fig. 2, when $V = 0$, $F/\Delta F = 0.55$, the $F/\Delta F$ ratio is not as very much greater than the approximately negative value of the coefficient, at very high values of $F/\Delta F$ where the number of revolutions per minute becomes very small. With such a propeller, the maximum blade-effect coefficient would therefore not be as very much

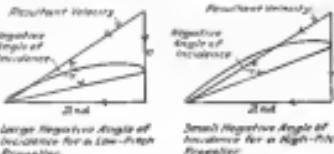


FIG. 3

greater than if the propeller were not revolving, and thus supporting forward motion only by virtue of its reaction area. The blade effect coefficient of a propeller is, in addition, in the case of curves of course, in some cases as great as that of propeller No. 5, and at an airspeed of 122 m.p.h., a $F/\Delta F$ ratio of 1.0 and engine speed of 115 r.p.m., would, with the same propeller diameter of 10 ft, give a negative thrust of 480 lb., which would check the landing run very seriously. Moreover, the negative thrust coefficient is not constant, nor greater than the values shown on the curves for very large values of $F/\Delta F$ and approaching the conditions of a nonrevolving propeller.

The investigation of Report No. 36 indicates that for a given propeller, the propeller with a larger blade effect than average blade, and that the blade effect of the high-pitch propeller is less than that of the lower-pitch propeller. The latter result is quite reasonable as can be seen from the diagramsmatic representation in Fig. 3, which with a low-pitch propeller the tendency will be for the air to strike the propeller blades at a larger angle in the direction of flight.

An airplane would be able to take off with a negative $F/\Delta F$ ratio if V decreased, and the $F/\Delta F$ ratio would decrease so that the propeller would run the propeller into a region of low blade-effect coefficient, or even of positive thrust, and at a positive angle of attack, which would be the result of a high-pitch propeller, it seems quite conceivable that propeller would find an advantage in experiencing an earlier landing run and then find the best possible conformation. The shortening of the landing run resulting therefrom might make all the difference between a poor and a good commercial machine.

Device for Shortening Landing Run

The most obvious device for shortening a landing run is the use of an airbrake consisting of flat plates hinged at the rear of the fuselage. To avoid the effect which is required to operate such a device, it is advisable to have the plates arranged so that they remain closed until the landing gear is down, while the opening of the under flap is retarded by the use of a fairly simple arrangement for the control of such an airbrake as shown in Fig. 4. The type of brak has been used on British machines, but obviously can be used to advantage on the grounds of simplicity and also the importance of the broken diaphragm as the speed decreases.

In a British test, the model body RR-1, shown in Fig. 5, had

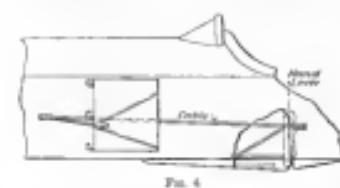


FIG. 4

a length of 200 ft in 47 one-tenths of the full run, and a length over of approximately 63 ft in 15. In a 60-ft per cent run, the increase in drift due to a 10-deg change of the landing angle is 0.000475, giving a landing coefficient of 0.003 per cent per 10 ft per cent change in landing angle, which will offer protection against the extremes for flat plates freely captioned.

The full one-tenth of the length for the RR-1 is 11.0 ft. Imagine length of 30-ft 1 ft runs applied to the Curtiss H-4-L machine, when landing run has been previously discussed and when in a single machine, the landing coefficient is 0.003 per cent per 10 ft, and the L-12 runs become 6.0 ft, natural of the 35 ft previously computed. It is quite clear from this that the effect of airbrake is small, and hardly worth reducing. On a very small machine it is possible that loss of a large area relative to the wing area might be offset by more satisfactory results.

While airbrakes may be set aside as entirely uninteresting, the use of wing flaps extending over almost the entire trailing

single-seat seaplane, and in a seaplane built by the Pringle Aviation Co. A diagrammatic scheme for a proposed seaplane is given in Fig. 6, showing the location of the flaps on the wings.

In 1936 British experiments carried out on an R.A.F. wing section with the largest zero-pitch moment increasing at 0.003 at the start, the maximum lift obtainable was increased in the ratio of 1.1 to 1.2, and the lift coefficient was increased by 0.001. The maximum lift occurs when the angle of incidence of the wing proper is of 6 to 8 deg. Each 10 deg increase in the maximum lift would decrease the landing speed in the ratio of 1 to 0.88. If the flaps were used individually by the pilot, it is felt that a 10-deg angle of incidence, after the flaps had been set at a spanwise angle corresponding to the maximum lift, by being an 18 to 20-deg pitch of 90 deg to the wing, with the wing itself at an angle corresponding to the three-point landing. Even when the flap was set at a 45-deg pitch angle to the wing, the drift of the aircraft was 0.000475, giving a landing coefficient of 0.003 per cent.

If the mechanism were applied to the Curtiss H-4-L machine and used as already explained, with flap set at a positive 10-deg angle for landing and a positive 60-deg angle for landing, the landing speed would become 300 m.p.h., and the landing coefficient would be about 0.001 per cent, which is 100 ft less than the 350 ft previously computed. A definite advantage would be that wing flaps extending along the rear edge would be of very appreciable help.

It was often assumed that a variable-pitch propeller will

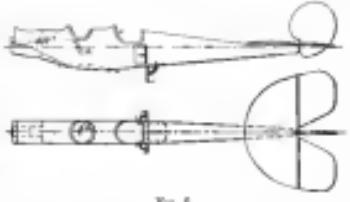


FIG. 5

decrease the landing speed by increasing the large negative drag of the propeller, but the effect of the propeller on the landing speed will indicate that this assumption is erroneous.

The small angles of pitch the effect of a variable-pitch propeller will be simply to increase the angle of glide for any speed of glide of the machine. It is only at a very steep, almost vertical, angle that the negative thrust will not affect the landing speed. The terminal velocity, the resistance due to the air resistance, and the effect of the propeller on the landing speed, which will always be defined by the maximum K_2 of the plane. The variable-pitch propeller will, however, have a very powerful effect reduced on the landing gear and will be probably the most efficient method of landing possible.

In Fig. 7, the forward skid experiments for the English Avro 615 the Avro aircraft are shown. Such arrangements are equivalent to the placing of an open tailfin and an extra rudder effect, without the airbrake effect, resulting in moving the center of gravity far behind the wheels, which means reduction in pitch-up and a dangerous effect on the landing gear. In the Avro 615, the skid tractive effect of 140 lb. were increased to 260 lb. by the use of a forward skid, the completed run would be shortened to 615 ft. instead of 769 ft., which is an appreciable difference.

Runns on wheels have been used without benefit of the pitch-up induced by moving the machine over, but with a small forward wheel or skid they might become a very useful device.

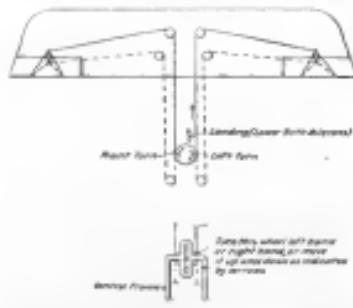


FIG. 6

very high values of $F/\Delta F$. Suppose we consider 30 m.p.h. the landing speed of the H-4-L machine. The negative thrust is given by the formula

$$(\text{Brake-effect coefficient}) \Delta F^2/180$$

and its maximum value occurs at a value of $F/\Delta F$ of approximately 1.8 when the brake-effect coefficient has a maximum value of 0.60. The landing speed for this value of $F/\Delta F$ must be 220 m.p.h. and the thrust will be

$$0.49 \times (0.60) \times (760) \times 0.060 = 187 \text{ lb.}$$

No power curves are given for the test and the question might be made is that to produce this negative thrust, the Liberty engine would have to develop more power than it can develop at 300 m.p.h. The negative thrust is then developed in only

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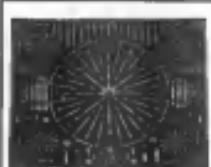
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